





SAM Webinars 2017: Overview of New Industrial Process Heat and CSP Capabilities in SAM 2017.1.17

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May 17, 2017

NREL is a national laboratory of the U.S. Department of Energy, Office of Energy Efficiency and Renewable Energy, operated by the Alliance for Sustainable Energy, LLC.

- Overview of New Industrial Process Heat and CSP Capabilities, May 17
- Modeling Molten Salt Power Tower Systems, May 18
- Electricity Rates and Monthly Bill Savings for Residential and Commercial Projects, June 1
- Modeling PV-Battery Systems, July 13
- Sizing Photovoltaic Systems, August 10
- SAM Open Source, September 21

Registration Links and Webinar Recordings

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Webinars	s and Rour	nd Tables	s								
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SAM webinars a				essions that cover vari	ous topics about \$	SAM. All webinars	are free and st	tart at 2 pm I	Mountain. To)	I
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Round Ta	bles										
SAM round table	es are 30-minute i	informal discus	sions online with	the SAM team.							
 Round table 	e registration (Jan	uary - June 201	17): Free, every o	other Thursday at 2:30	pm Mountain Tim	e.					
		Mohinara									
Recording	ys of Fast (vvenilars									
	-			following webinars.							

- Modeling a Photovoltaic Battery System in SAM 2016.3.14
- Modeling a Residential Photovoltaic System in SAM 2016.3.14
- SAM Demonstration in Spanish, June 2016

2015 Webinars

Battery Storage for Photovoltaic Systems, Sep 2015

https://sam.nrel.gov/webinars

- What's new for CSP in SAM 2017.1.17
- Industrial process heat (IPH) applications
- Levelized cost of heat
- Parabolic trough IPH model
- Linear direct steam IPH model
- Q&A

- CSP Generic model uses a different set of regression equations for thermal losses, power cycle conversion efficiency, and parasitic consumption
 - You will need to calculate different coefficients than for the CSP Generic model in older versions of SAM
- Power Tower model improvements to dispatch and solar field optimization algorithms
 - Tomorrow's webinar will discuss in detail
- New solar industrial process heat models

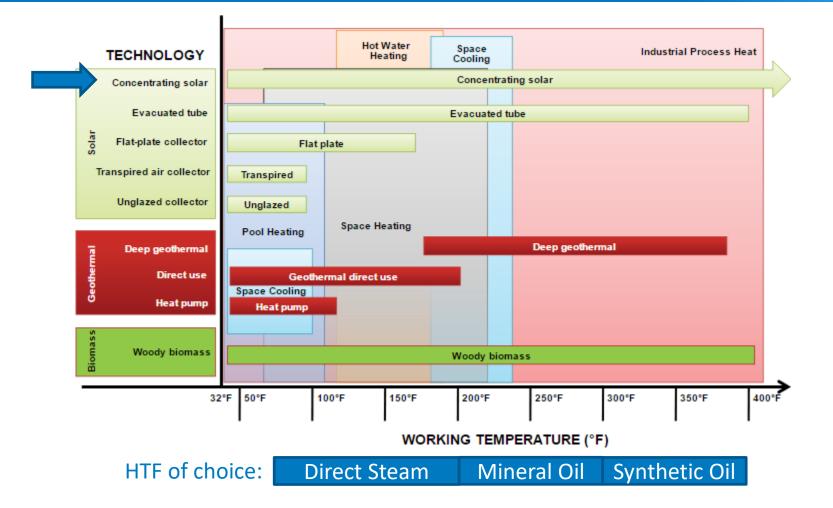
- Concentrating solar power (CSP)
 - A field of concentrating solar collectors coupled to a power cycle for electricity generation. Typically hightemperature, large-scale systems.
- Solar industrial process heat (SIPH, or IPH for short)
 - Concentrating solar collectors used to provide direct team or heat for a thermal process.
 - Typically replaces or supplements a fossil-fired boiler





IPH = Process Heat

IPH applications for trough and linear Fresnel collectors



Source: EPA Renewable Heating and Cooling website, Kurup (2015) paper on IPH for Southwest U.S.

Overview of market status for IPH prepared by NREL for DOE

- Applications
- Collector costs
- Examples of systems
- References to other sources of information

www.nrel.gov/docs/fy16osti/64709.pdf



Initial Investigation into the Potential of CSP Industrial Process Heat for the Southwest United States

Parthiv Kurup and Craig Turchi National Renewable Energy Laboratory

NREL is a national laboratory of the U.S. Department of Energy Office of Energy Efficiency & Renewable Energy Operated by the Alliance for Sustainable Energy, LLC

This report is available at no cost from the National Renewable Energy Laboratory (NREL) at www.nrel.gov/publications.

Technical Report NREL/TP-6A20-64709 November 2015

Contract No. DE-AC36-08GO28308

 $LCOH = \frac{(Total \ installed \ project \ cost) * (FCR) + (Annual \ O\&M)}{Annual \ thermal \ generation}$

- Accounts for installation and operating costs, project financial requirements, and thermal output of collectors
- Compare cost per kWh of solar-thermal energy to cost per MMBtu of natural gas
- Not a cash-flow method
- Requires calculating the fixed charge rate, which is the revenue per amount of investment required to cover the investment cost
 - Described in SAM's "LCOE Calculator" Help topic

IPH models in SAM were adapted from CSP models

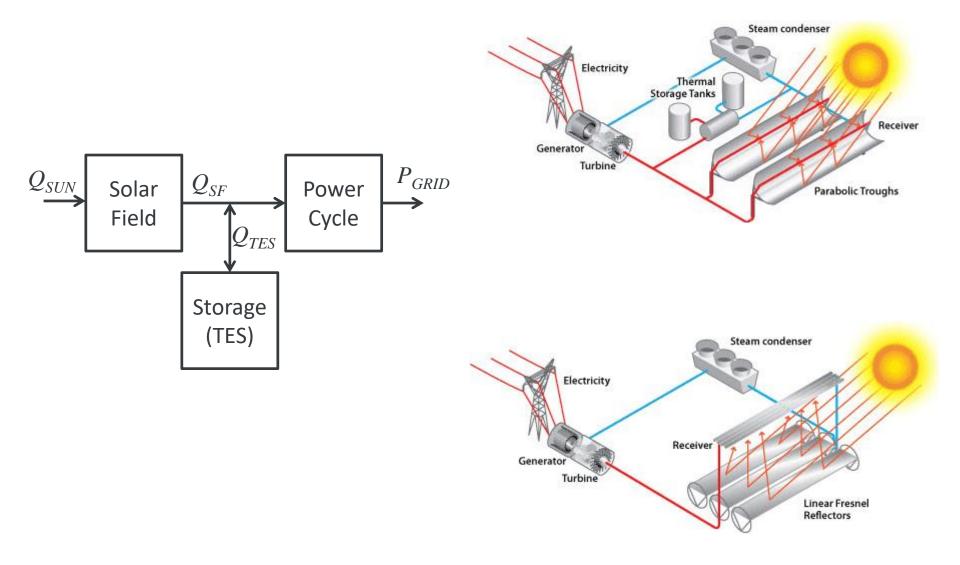
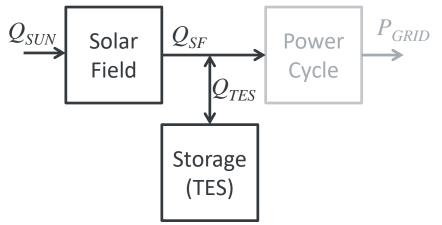


Image source DOE EERE

Before SAM 2017.1.17, model IPH by ignoring power cycle



Power cycle capacity equal to solar field capacity

Boiler operating pressure to saturated steam pressure at solar field outlet temperature

Auxiliary heater outlet temperature to solar field outlet temperature

Power cycle startup time and power to zero, startup temperature to field inlet temperature

Max overdesign operation to 2 times design, and minimum operation to 0.02

Cost and financial models not designed for thermal application

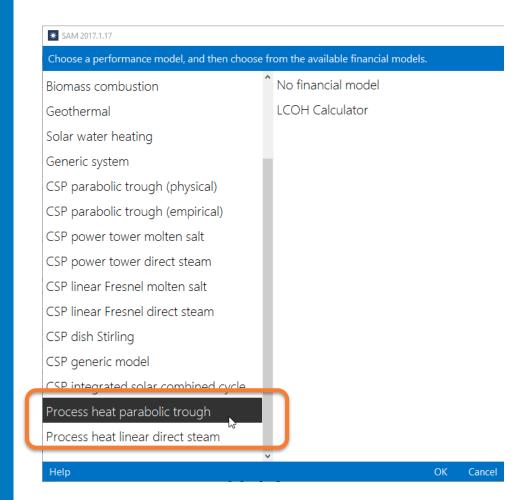
Technique described in 2015 milestone report to DOE, available on SAM website: "Geothermal Risk Reduction via Geothermal/Solar Hybrid Power Plants"

- IPH parabolic trough
 - Physical model of trough collectors and receivers
 - Pressurized water, oil, or salt HTF
- IPH linear direct steam
 - General optical model of field
 - Most of receiver at a single temperature
 - Saturated steam with user-specified steam quality
 - Two-phase steam at field outlet, completely condensed at inlet

- The IPH trough model may be a better starting point because it characterizes the system performance more completely
 - $_{\odot}\,$ Physical models of collector and receiver
 - Calculate pressure drops
 - Flow rate limitations
- IPH direct steam model requires more data as input
 - Optical efficiency tables or IAM coefficients
 - Pressure drops

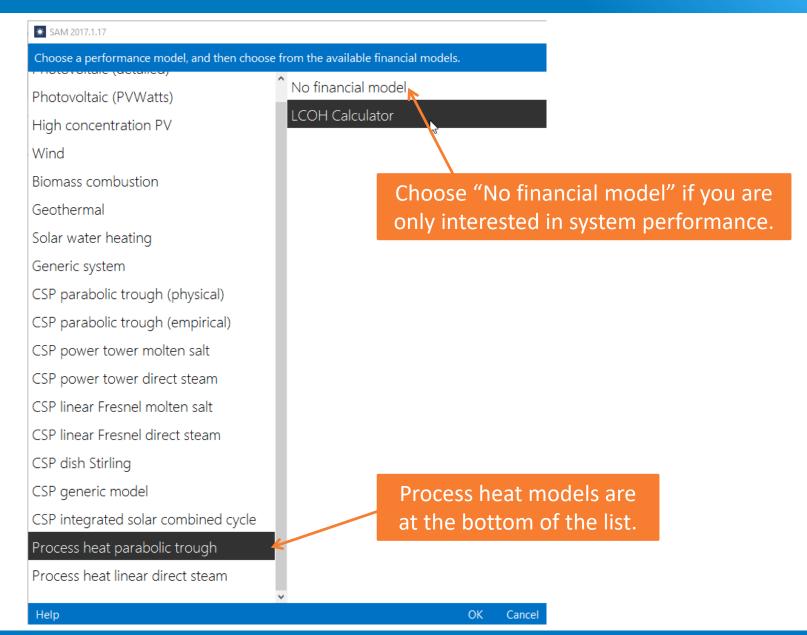
IPH models in SAM 2017.1.17

- Physical trough and linear direct steam models with power cycle removed
- Basic financial model uses fixed-charge-rate method to calculate "levelized cost of heat"
- Thermal storage not yet implemented, coming soon! (for IPH trough with oil or salt HTF only)



IPH Parabolic Trough

Choose performance model and financial model



Input pages provide access to input parameters

* SAM 2017.1.17		- 🗆 X
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IPH Trough, LCOH Calculator	Download a weather file from the NREL NSRDB	
Location and Resource	Click Download and type a street address o Oownload Click Download and type a street address o weather file from the NREL NSRDB for Unite	
System Design	SAM adds the downloaded file to the solar Opens the Help system	
	NSRDB Map	
Solar Field	Choose a weather file from the solar resource library	
Collectors (SCAs)	Click a name in the list to choose a file from the library. Type a few letters of the name in the search box to filter the list. If your location is not in the downloading a file (see above).	library, try
Receivers (HCEs)	Search for: Name ~	
Financial Parameters	Name Station ID Latitude Longitude Time zone E	ilevation ^
K	USA CA Fresno (TMV2) 93193 36.7667 -119.717 -8 1	00
		02
	55001 11505 0 2	.9
		4
	USA UISPIAY IIIPUL PASCS. 32.833 -115.583 -8 -1	17
		>
	City Imperial Time zone GMT -8 Latitude 32.833 °N	
		ta
	State CA Elevation -17 m Longitude -115.583 °E Refresh lib	orary
	Country USA Data Source TMY3 Station ID 747185 Folder setti	ings
	Data file C:\SAM\2017.1.17\solar_resource\USA CA Imperial (TMY3).csv Open library	folder
	-Annual Weather Data Summary	
	Global horizontal 5.77 kWh/m²/day Average temperature 23.4 °C	
	Click ((Circulate)) to wrage wind speed 3.3 m/s	
	Click "Simulate" to run a	vebsite
	simulation.	
Simulate > Parametrics Stochastic	Check the bow and click E Supported solar weather running a simulation.	Browse
P50 / P90 Macros		

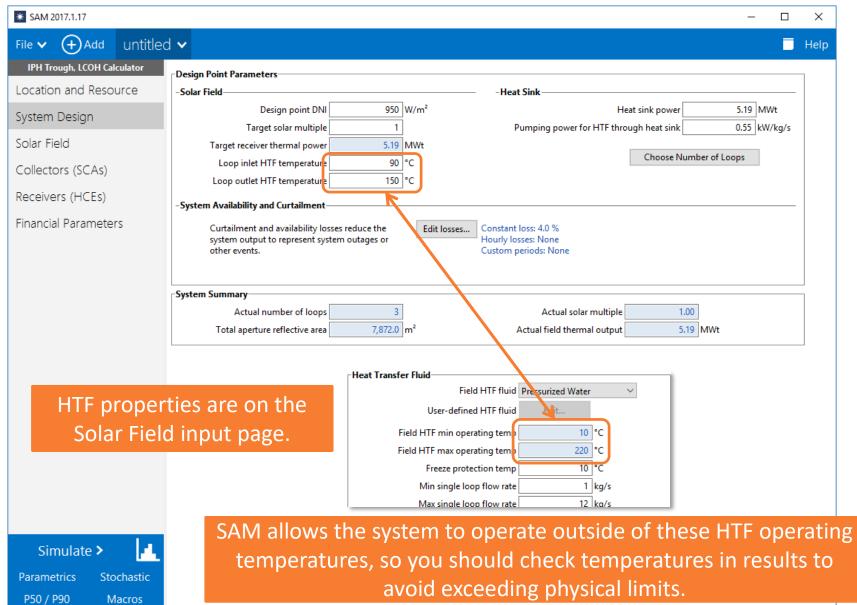
Weather file provides information about solar resource and ambient conditions

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File 🗸 🕂 Add untitled	- -	Download	weathe	er data		Ē	Help
Generic CSP, Single owner	□ Download a weather file from the NREL NSKDB-	from t	he NSRI	ר R			
Location and Resource	Click Download				AM's CSP mode		
Solar Field	wedenet me from	the NREL NSRDB for United St wnloaded file to the solar reso			than the NREL NS details.		
Power Block							
Thermal Storage	Choose a weather file from the solar resource library. Click a name in the list to choose a file from the library. downloading a file (see above).	,	e in the search box to	o filter the list. If yo	ur location is not in	the library, try	
System Costs							
Lifetime	Search for: Name ~						
Lieune	Name	Station ID	Latitude	Longitude	Time zone	Elevation	^
Financial Parameters	USA CA Chino Airport (TMY3)	722899	33.967	-117.633	-8	198	
	USA CA Chula Vista Brown Field Naas (TMY3)	722904	32.583	-116.983	-8	159	
Time of Delivery Factors	USA CA Concord Concord-buchanan Fiel (TMY3)	724936	38	-122.05	-8	7	
In continues	USA CA Crescent City Faa Ai (TMY3)	725946	41.783	-124.233	-8	17	
Incentives	USA CA Daggett (TMY2) USA CA Daggett Barstow-daggett Ap (TMV3)	23161	34.8667	-116.783	-8	588	~
Depreciation	<	773815	24.85	-116.8	-*	585	•
	C': Depart	Time zone GMT -8		34.8667 °N	-Tools		
	City Daggett	Time zone GMT -8	Latitude	54.0007 N	View	data	
	State CA	the file ster		-116.783 °E	Refres	h library	
	Country Cook	ther file sto	red on	20161		settings	
	Data file C:\SAM\2017.1.17	computer.			Open lib	rary folder	
	-Annual Weather Data Summary				_		
	Global horizontal 5.86 kWh	h/m²/day Average ter	mperature	19.8 °C			
	Direct normal (beam) 7.65 kWh	h/m²/day Average w	ind speed	4.9 m/s			
	Diffuse horizontal 1.34 kWh	h/m²/day		Ţ	/isit SAM weather d	ata website	
	Use a specific weather file on disk						
Simulate >						Browse	
Parametrics Stochastic	Check the box and click Browse to choose a weather fi Supported solar weather file formats are SAM CSV, TM		ithout adding it to t	he solar resource lib	orary.		
P50 / P90 Macros		- *					

System Design page provides access to main design parameters

* SAM 2017.1.17		Decign	point DNI determines size of field
File 🗸 🕂 Add untitled	i •		ng summer solstice sun position.
IPH Trough, LCOH Calculator	Design Point Parameters	assann	ng summer solstice sun position.
Location and Resource	-Solar Field		
System Design	Design point DNI	950 W/m²	Target solar multiple for systems
System Design	Target solar multiple	1	
Solar Field	Target receiver thermal power	5.19 MWt	with storage (available soon).
Collectors (SCAs)	Loop inlet HTF temperature	90 °C	Choose Number of Loops
Collectors (SCAS)	Loop outlet HTF temperature	150 °C	
Receivers (HCEs)	-System Availability and Curtailment		
Financial Parameters	Curtailment and availability losses reduce the system output to represent system outages o other events.		Constant Ioss: 4.0 % Hourly Iosses: None Custom periods: None
	System Summary		
	Actual number of loops	3	Actual solar multiple 1.00
	Total aperture reflective area 7,8	72.0 m ²	Actual field thermal output 5.19 MWt
Simulate >			
Parametrics Stochastic			
P50 / P90 Macros			

Loop inlet and outlet temperatures should be within HTF operating ranges



Heat sink power is the capacity or thermal load of the system in thermal megawatts

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	Design Point Parameters -Heat Sink -Solar Field -Heat Sink Design point DNI 950 W/m² Target solar multiple 1 Target receiver thermal power 5.19 MWt Loop inlet HTF temperature 90 °C Loop outlet HTF temperature 90 °C Avalue for the heat sink power, or click "Choose Number of Loops	
Loops to	Calculate the power based on a desired number of loops. System Summary Actual number of loops 3 Total aperture reflective area 7,872.0 m² Actual field thermal output 5.19	
	SAM calculates the system summary parameters based on the other values you enter.	
Simulate > Parametrics Stochastic P50 / P90 Macros		

The "constant loss" of 4% is equivalent to a 96% annual availability factor and may not be appropriate for IPH

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IPH Trough, LCOH Calculator Location and Resource System Design Solar Field Collectors (SCAs) Receivers (HCEs) Financial Parameters	Design Point Parameters -Solar Field -Heat Sink Design point DNI 950 W/m² Heat sink power Target solar multiple 1 Pumping power for HTF through heat sink 0.55 KW/kg/s Target receiver thermal power 5.19 MWt Loop inlet HTF temperature 90 *C Choose Number of Loops Loop outlet HTF temperature 150 *C -System Availability and Curtailment Curtailment and availability losses reduce the Edit losses	
	e default value. I otal aperture reliective area 7,672.0 m ² Hourly losses: None Custom periods: None Cus	
Simulate >		
Parametrics Stochastic P50 / P90 Macros		

Solar Field inputs are the same as physical trough model, except for piping between solar field and heat sink

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IPH Trough, LCOH Calculator Location and Resource System Design Solar Field Collectors (SCAs) Receivers (HCEs)	System Design Parameters Design Point DNI 950 W/m² Loop inlet HTF temperature 90.0 °C Target solar multiple 1.00 Loop outlet HTF temperature 150.0 °C Target receiver thermal power 3.46 MWt Solar Field Design Point Single loop aperture 2,624.0 m² Actual number of loops 2 Loop optical efficiency 0.7213 Total aperture reflective area 5,248.0 m² Total loop conversion efficiency 0.6937 Actual solar multiple 1.00	
	Total required aperture, SM=1 5,248.0 m² Actual field thermal output 3,46 MWt Required number of loops, SM=1 2.00 4ctual field thermal output 3,46 MWt Solar Field Parameters 0 10 4ctual field thermal output 3,46 MWt Solar Field Parameters 0 15 m Field HTF fluid Freesurized Water Stow angle 170 deg 0 0 Edit Field HTF fluid Edit Field HTF fluid Edit Header pipe roughness 4.57e-005 m 10 °C Field HTF max operating temp 10 °C Field Allows al loss coefficient 0.45 W/m²-K N/m²-K N/m² - K <	ctior
Simulate > Parametrics Stochastic P50 / P90 Macros	Model piping through heat sink? Feader design min flow velocity 2 m/s Length of piping through heat sink 500 m Header design min flow velocity 2 m/s Collector Orientation These parameters affect heat loss, thermal inertia and capacity, and pumping power. Inertia and capacity, and pumping power. Mirror Washing Water usage per wash 0.7 L/m², aper. Plant Heat Capacity Washes per year 12 Hot piping thermal inertia 0.2 kWht/K-MWt	 Ŷ

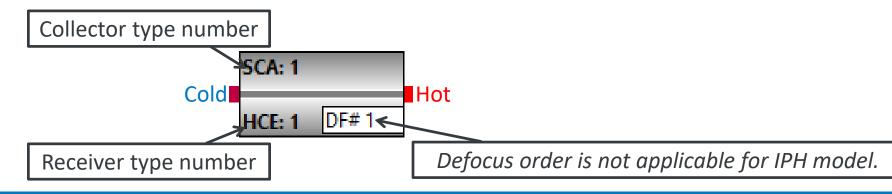
Unlike CSP model, IPH model only defocuses field when HTF flow rate exceeds maximum flow limit

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IPH Trough, LCOH Calculator	System Design Parameters	
Location and Resource	Design Point DNI 950 W/m ²	Loop inlet HTF temperature 90.0 °C
System Design	Target solar multiple 1.00 Target receiver thermal power 3.46 MWt	Loop outlet HTF temperature 150.0 °C
Solar Field	Target receiver thermal power 3.46 MWt	
Solar Field	Solar Field Design Point	
Collectors (SCAs)	Single loop aperture 2,624.0 m ²	Actual number of loops 2
Receivers (HCEs)	Loop optical efficiency 0.7213	Total aperture reflective area 5,248.0 m ²
Receivers (FICES)	Total loop conversion efficiency 0.6937	Actual solar multiple 1.00
Financial Parameters	Total required aperture, SM=1 5,248.0 m ²	Actual field thermal output 3.46 MWt
	Required number of loops, SM=1 2.00	
	Solar Field Parameters	Heat Transfer Fluid
	Row spacing 15 m	Field HTF fluid Pressurized Water 🗸 🗸
	Stow angle 170 deg	User-defined HTF fluid Edit
Flow	v rate limits determine when t	Field HTF min operating temp 10 °C
FIOW	rate mints determine when i	Field HTF max operating temp 220 °C
	defocuses collectors.	Freeze protection temp 10 °C
	Traching agreed as SCA 125.0 W/sea	Min single loop flow rate 1 kg/s
	Tracking power per SCA 125.0 W/sca Total tracking power 1,000.0 W	Max single loop flow rate 12 kg/s
	Number of field subsections 1	Min field flow velocity 0.228212 m/s
		Max field flow velocity 2.87905 m/s
	Model piping through heat sink?	Header design min flow velocity 2 m/s
	Length of piping through heat sink 50.0 m	Header design max flow velocity 3 m/s
	Collector Orientation	
	Collector tilt deg	Tilt: horizontal=0, vertical=90
	Collector azimuth 0 deg	Azimuth: equator=0, west=90, east=-90
Simulate >	Mirror Washing	Plant Heat Capacity
	Water usage per wash 0.7 L/m ² , aper.	Hot piping thermal inertia 0.2 kWht/K-MWt
Parametrics Stochastic		Cold piping thermal inertia 0.2 kWht/K-MWt
P50 / P90 Macros	Washes per year 12	Field loop piping thermal inertia 4.5 Wht/K-m

Collectors are the same as the CSP physical trough model

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	IPH Trough, LCO)H Calculator	Collector Library	1
Lo	ocation and F	Resource	Search for: Name ~	
Sy	/stem Design	I	Name Reflective aper Aperture width Length of colle Number of mo	
Sc	olar Field		AlbiasaTrough AT150 817.5 5.774 150 12 Siemens SunField 6 545 5.776 95.2 8	
C	ollectors (SC)	Δς)	SkyFuel SkyTrough (with 80-mm OD receiver) 656 6 115 8	
			FLABEG Ultimate Trough RP6 (with 89-mm OD receiver for oil 1720 Diagram of loop showing positio	n
Re	eceivers (HCI	ES)	Collector types in loop configuration Cold - 1 - 1 - 1 - Hot ← Of collector types as defined at	
Cho	oose a d	collect	or from the bottom of Solar Field page.	
		library.	ary SkyFuel SkyTrough (with 80-mm OD receiver) Apply Values from Library	
			Reflective aperture area and then apply parameters from library to collector type. Optical Parameters Incidence angle modifier coefficients Edit data Geometry effects 0.952 Mirror reflectance 0.933 General optical error 1 Dit on mirror Optical Calculations Ength of single module 14.375 m Length of single module 1.00176 Optical efficiency at design Optical Calculations Incidence angle module 0.9264 Length of single module 1.00176 0.90164 Optical Calculations 0.902614 0.902614 Length of single module 1.00176 0.902614 Optical efficiency at design 0.848494	
	Simulate Parametrics P50 / P90	Stochastic Macros	 ← Collector Type 3 Collector Type 4 For the field contains different types of collectors, configure them here. 	

Use the loop configuration to set number of collectors per loop, assign collector and receiver types, and defocus order Loop configuration is at bottom of Solar Field page. You may need to scroll down the window to see it. 1. Type number of collector Single Loop Configuration assemblies per loop. The specification below is on 3. Choose an item to edit. Usage tip: To configure the loop, choose whether to edit SCAs, HCEs or defocus order. Sele multiple items. Assign types to selected items by pressing keys 1-4. Number of SCA/HCE assemblies per loop: 4 SCAs Edit HCEs Catit Defocus Order Reset Defocus SCA: 1 SCA: 1 SCA: 1 SCA: 1 DF# 4 DF# 3 DF# 2 DF# 1 HCE: 1 HCE: 1 HCE: 1 HCE: 1 4. Type a number on your 2. Click a collector to select it. keyboard to change a number.



Receivers are the same as the CSP physical trough model

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File 🗸 🕂 Add untitled	v							Help
IPH Trough, LCOH Calculator	Receiver Library							
Location and Resource	Search for: Na	me		~				
System Design	Name Solel UVAC 3		Absorber tube 0.066	Absorber tube 0.07	Glass envelope 0.115	Glass envelope 0.121	^	
Solar Field	Siemens UVAC 2010		0.066	0.07	0.109	0.115		
Collectors (SCAs)	Schott PTR80 Roval Tech CSP RTUVR 2014 (Manufac <	cturer Specifications)	0.076 0.066	0.08 0.07	0.115 0.119	0.12	~	
Receivers (HCEs)	Receiver types in loop configura	ation Cold - 1 - 1 - 1 - 1	- Hot					
Financial Parameters	Receiver Type 1							
	Receiver name from library Schot	t PTR80			Apply Values from	n Library		
	Receiver Geometry							
	Absorber tube inner	diameter 0.0	76 m Ał	bsorber flow plug di	ameter	0 m		
	Absorber tube outer	diameter 0.	08 m	Internal surface rou	ghness 4	.5e-005		
	Glass envelope inner	diameter 0.1	15 m	Absorber flow	pattern Tube flow	\sim		
	Glass envelope outer	diameter 0.	12 m	Absorber mater	ial type 304L	\sim		
	Parameters and Variations							
		Variation 1	Variation 2	Variation 3	Variation 4	4*		
	Variant weighting fraction*	0.985	0.01	0.00	5	0		- 1
	Absorber Parameters:							
	Absorber absorptance	0.963	0.963	0.	8	0		
	Absorber emittance	Table Table	Table 0.65	Table 0.6	5 Table	0		
	Envelope Parameters:							
	Envelope absorptance		0.2		0	0		
	Envelope emittance		0.86		1	0		
Simulate >	Envelope transmittance		0.964 Broken Glass	Broken Glass	1 Broken	0 Glass		
				E broken oldss		0.033		
Parametrics Stochastic	Gas Parameters: Annulus gas type	Δir	Air ~	Air	~ Air	~		
P50 / P90 Macros	Annulus pressure (torr)		750	75	_	0		

Either provide a fixed charge rate (FCR), or use the calculator and provide financial parameters

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IPH Trough, LCOH Calculator	Levelized Cost Of Heat
Location and Resource	This model calculates the levelized cost of the net heat delivered from the solar field to the heat sink. The solar field model consumes electricity to
System Design	pump the HTF and move the collectors. The model applies the electricity rate specified here to calculate the annual electricity cost. This cost is then added to the specified fixed annual operating cost in the finanical model. The Levelized Cost of Energy Calculator structure below is taken unchanged from the other SAM models, but the user should note that here it estimates levelized cost of thermal energy (LCOH).
Solar Field	· · · · · · · · · · · · · · · · · · ·
Collectors (SCAs)	Electricity Rate 0.060 \$/kWh
Receivers (HCEs)	LCOE Calculator
Financial Parameters	The fixed-charge rate method of calculating the levelized cost of energy simplifies time-dependent calculations and is appropriate for market-level analysis such as for the NREL Annual Technology Baseline, or for very preliminary project analysis. The cash flow method of SAM's other financial models is more suitable for more detailed project analysis. See Help for details.
	NREL Annual Technology Baseline and Standard Scenarios website
	-Capital and Operating Costs FCR is either the value you enter,
	System capacity 3,458.61 kW the value SAM calculates from th
	O Enter costs in \$ O Enter costs in \$/kW Capital cost 2755 000.00 560.00 financial parameters you enter.
	Capital cost 2,755,000.00 560.00 560.00
	Fixed operating cost (annual) 41,600.00 8.00
	Variable operating cost 0.0010 \$/kWh
	-Financial Assumptions O Enter fixed charge rate
	Fixed charge rate (real) 1 Analysis period 10 years Fixed charge rate (FCR) 0.108
	Inflation rate 2.5 %/year FCR = CRF · PFF · CFF (see below)
	Internal rate of return (nominal) 0 %/year
	Project term debt 100 % of capital cost
	Nominal debt interest rate 5 %/year
	Effective tax rate 40 %/year
Simulate >	Depreciation schedule Edit % of capital cost
	Annual cost during construction 100 % of capital cost
Parametrics Stochastic	Nominal construction interest rate 0 %/year
P50 / P90 Macros	-Reference Values V



When you run a simulation, SAM performs a set of calculations for each hour of the year to calculate the thermal energy produced by the solar field.

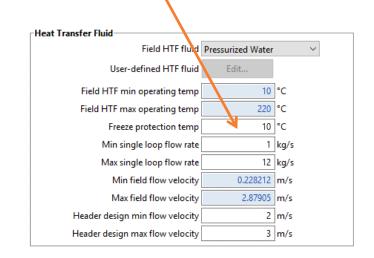
If you included the LCOH calculator when you created the case, it also calculates the LCOH using the sum of the hourly energy values to represent the total annual thermal energy produced by the field.

The Metrics table displays a summary of results

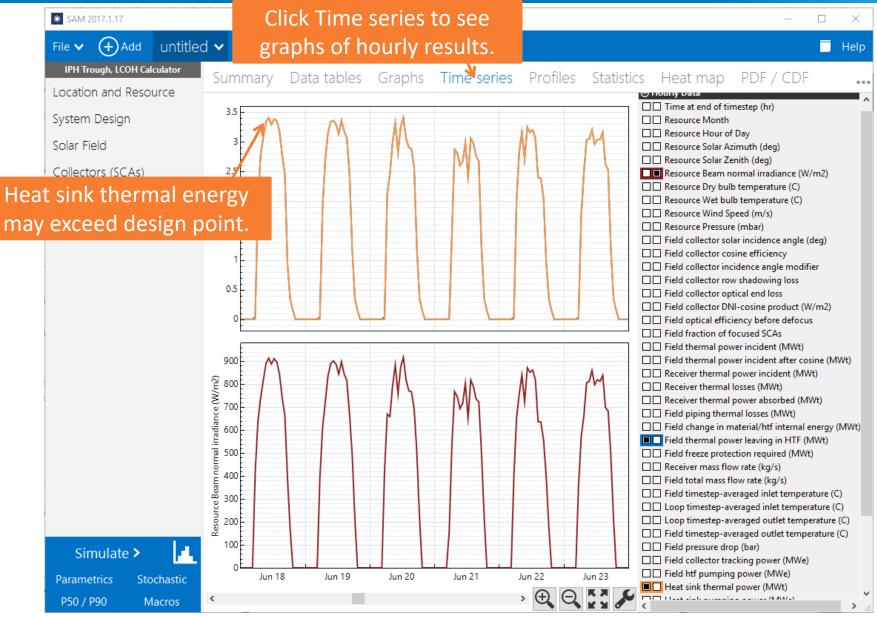
- Annual energy and annual field energy are the same when freeze protection energy is zero
- Electricity load is for pumping and tracking power

Metric	Value
Annual energy (year 1)	7,389,923 kWh-t
Annual field energy (year 1)	7,389,923 kWh-t
Annual thermal freeze protection (year 1)	0 kWh-t
Annual electricity load (year 1)	25,762 kWh-e
Levelized cost of heat	3.33 ¢/kWh-t

Increase freeze protection temperature on Solar Field page to add freeze protection energy.



With no freeze protection, field thermal power is the same as heat sink thermal power



- 1. Change HTF on Solar Field page
 - HTF operating temperatures change with the HTF
 - For a real analysis, you would need to change the costs to account for the different HTF
- 2. On the System Design page, change design loop inlet and outlet temperature to be consistent with oil HTF
- 3. On the Solar Field page, change the freeze protection temperature to 220 °C
- 4. Run a simulation, and note the differences in results
 - Field and heat sink energy are different
 - Heat loss and pumping power changes with HTF

IPH Linear Direct Steam

For IPH direct steam, you do not configure the loop

SAM 2017.1.17: F:\OneDrive\SAM\I	PH\2017 Webinar\webinars-2017-industrial-process-heat-example.sam]
File 🗸 🕂 Add Trough	🗸 Linear Direct 🗸		🗌 Help	
IPH Linear (steam), LCOH Calculator Location and Resource System Design Solar Field Collector and Receiver Financial Parameters	Design Point Parameters -Solar Field -Heat Sink Design point DNI 950 W/m² Target solar multiple 1.2 Target receiver thermal power 6.00 MWt Field inlet temperature 100 °C Field outlet steam quality 0.75 -System Availability and Curtailment October 100			and
	Curtailment and availability losses reduce the system output to represent system outages or other events. Fault loss may not be ate for an IPH system.			
Simulate >	Direct steam model requires knowledge of pred drops versus size and number of modules in a unlike the trough model, they are not calculat model. Also, no flow rate limits.	loop –	ne	

Solar field consists of a single boiler section with no superheater

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IPH Linear (steam), LCOH Calculator	System Configuration		
Location and Resource	Design Point DNI	950 W/m²	Field inlet temperature 100.0 'C
Suctom Docian	Target solar multiple	1.20	Heat sink inlet pressure 20.0 bar
System Design	Target receiver thermal power	6.00 MWt	Field outlet steam quality 0.75
Solar Field	Solar Field Design Point		
Collector and Receiver	Single loop aperture	3081.6 m2	Actual number of loops 2
Financial Descentes	Loop optical efficiency	0.74613	Actual aperture 6163.2 m2
Financial Parameters		.977567	Actual solar multiple 0.854124
		0.729392	Actual field thermal output 4.27062 MWt
		7215.81 m2	
	Required number of loops, SM=1	3	
	Solar Field Parameters		Steam Design Conditions
	Number of modules in boiler section	S	et number of boiler modules to set the
	Solar elevation for collector nighttime stow		
	Solar elevation for collector morning deploy	10 deg	single loop aperture area.
	Stow wind speed Collector azimuth angle	20 m/s	Freeze protection temperature 10 'C
	Design point ambient temperature	42 'C	Field pump efficiency 0.85
	Tracking power	0.20 W/m2	
	Piping thermal loss coefficient	0.0035 W/K-m2-aper	
	Mirror Washing Design ambie	nt temner	ature used in
	heat loss po	lynomials	to calculate
	Land Area thermal e	efficiency a	at design
	Solar field area	children cy c	Total land area 1.82755 acres
Simulate > 🗾 📕			
Parametrics Stochastic			
P50 / P90 Macros			
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Pressure drops not modeled physically

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IPH Linear (steam), LCOH Calculator	System Configuration			
Location and Resource	Design Point DNI	950 W/m²	Field inlet temperature 100.0 'C	
System Decian	Target solar multiple	1.20	Heat sink inlet pressure 20.0 bar	
System Design	Target receiver thermal power	6.00 MWt	Field outlet steam quality 0.75	
Solar Field	Solar Field Design Point			
Collector and Receiver	Single loop aperture	3081.6 m2	Actual number of loops 2	
Financial Parameters	Loop optical efficiency	0.74613	Actual aperture 6163.2 m2	
Financial Parameters	Loop thermal efficiency	0.977567	Actual solar multiple 0.854124	
	Total loop conversion efficiency	0.729392	Actual field thermal output 4.27062 MWt	
	Total required aperture, SM=1	7215.81 m2		
	Required number of loops, SM=1	3		
	Solar Field Parameters		Steam Design Conditions	_]
	Number of modules in boiler section	6	Cold header pressure drop fraction 0.01	=
	Solar elevation for collector nighttime stow	10 deg	Boiler pressure drop fraction 0.075	= 1
	Solar elevation for collector morning deploy	10 deg	Average design point hot header pressure drop fraction 0.025	=
	Stow wind speed	20 m/s		2 bar 0 'C
	Collector azimuth angle Design point ambient temperature	0 deg 42 'C	Freeze protection temperature 10 Field pump effigiency 0.85	=
	Tracking power	42 C	Field pump endency 0.0	<u> </u>
	Piping then	0.20		
	Mirror Washing Pressu	re drops are d	o not scale with size of loop.	
	wasnes per year	12		
	Land Area			
	Solar field area 1.52296 acre	Non-solar field land area	multiplier 1.2 Total land area 1.82755 acres	
Simulate >				
Parametrics Stochastic				
P50 / P90 Macros	<			>

BUG: Actual number of loops is not calculated correctly

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IPH Linear (steam), LCOH Calculator	∑System Configuration	
Location and Resource	Design Point DNI 950 W/m² Field inlet temperature 100.0 'C	
System Design	Target solar multiple 1.20 Heat sink inlet pressure 20.0	
	Target receiver thermal power 6.00 MWt Field outlet steam quality 0.75	
Solar Field	Solar Field Design Point	
Collector and Receiver	Single loop aperture 3081.6 m2 Actual number of loops 2	
Financial Parameters	Loop optical efficiency 0.74613 Actual aperture 6163.2 m2	
	Loop thermal efficiency 0.977567 Actual solar multiple 0.854124 Total loop conversion efficiency 0.729392 Actual field thermal output 4.27062 MWt	
	Total required aperture, SM=1 7215.81 m2	
	Required number of loops, SM=1 3	
	Bug in the user interface calculation results in a field hop fraction	0.01
		0.075
	with one fewer loop than indicated here. For IPH	0.025
	systems with small fields, this can significantly decrease ressure drop	2.2 bar
	the color multiple	10 'C
		0.85
	WILL BE FIXED IN UPDATE AT END OF MAY 2017	
	∩Mirror Washing	
	Water usage per wash 0.02 L/m2 ap	
	Water usage per wash 0.02 L/m2, ap Washes per year 12 Thermal inertia per unit area of solar field 2.7	′K-m2
	Land Area	
	Solar field area 1.52296 acre Non-solar field land area multiplier 1.2 Total land area 1.82755 acr	es
Simulate >		
Parametrics Stochastic		
P50 / P90 Macros		>
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Boiler section is a generic optical model, and requires that you characterize the field optical efficiency outside of SAM

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IPH Linear (steam)	LCOH Calculator												^
Location and		-Boiler Geometry a		erformance— tive aperture ar	ea	513.6 m2						1	
System Desig	n		Length of	collector modu	ule	44.8 m							
Solar Field				Tracking en Geometry effeo		1	Optical c	haracterization	method:				
Collector and	Receiver			Mirror reflectiv	ity	0.935		position table					
Financial Para	meters			Mirror soili neral optical err	ng	0.95		ector incidence a lence angle mod	-				
		-Solar Position/	Collector In	cidence Angle	Table								
		Import	Ехро	ort	Сору	Paste	Rows (zenith):		11 Cols (azim	nuth):	20		
		0	-180	-160	-140	-120	-100	-80	-60	-40	^		
		0	1	1	1	1	1	1	1	1			
		10	0.98	0.974445	0.971976	0.972847	0.97691	0.97691	0.972847	0.971976	i i		
		20	0.93	0.922976	0.92893	0.946005	0.954019	0.954019	0.946005	0.92893			
		30	0.84	0.838618	0.870691	0.913021	0.940911	0.940911	0.913021	0.870691			
		40	0.72	0.729947	0.803687	0.866961	0.900039	0.900039	0.866961	0.803687			
		50	0.55	0.591255	0.707454	0.793509	0.83956	0.83956	0.793509	0.707454			
		60	0.34	0.432178	0.597478	0.664006	0.693511	0.693511	0.664006	0.597478			
		70 80	0.13	0.265254	0.425586	0.464496	0.477106	0.477106	0.464496	0.425586			
		<	0.01	0.113094	0.20891	0.233255	0.238828	0.238828	0.233200	0.20891	× .		_
		Specifying sola Specifying coll angles (deg)	ector incider	ce angle table:	ate solar zenith Rows indicate I	long						M coefficient h literature.	ts
		-Incidence Ang	gle Modifier	Coefficients-		Const	C1	C2 (C3 (54			
		.	Transverse ir	icidence angle	modifier	0.9896		.0721 -0.2		0			
				icidence angle		1.0031		.5368 -1.6					
			-	-	mounter	1.0051	-0.2239 0	.5500 -1.0	434 0.72			j	
Simulate	e> 🛃	Receiver Geomet	ry and Heat	Loss	Polynomia	al heat loss mo	del	~]	
Parametrics	Stochastic	-Polynomial fit	heat loss mo	del									
P50 / P90	Macros	_		<u></u>) (W/m)	C1 (W/m-K)	C2 (W/m-K/			//m-K^4)			
1307130	Wacros	Steam	emperature	adjustment	0	0.6	72 0.002	2556	0	0			×

Receiver parameters

IPH Linear Getam), LCOH Calculator Location and Resource System Design Solar Field Collector and Receiver Financial Parameters Absorber tube inner diameter Ologe outer diameter Olass envelope Olass Olass	× Help
IPH Linear (steam), LCOH Calculator Location and Resource System Design Solar Field Collector and Receiver Financial Parameters Absorber tube inner diameter O.002 Glass envelope outer diameter O.0112 m Absorber flow plug diameter O.02 Variation 1 Variation 2 Variation 3 Variation 4* Variation 1 Variation 2 Variation 3 O.005 O.005	Help
Location and Resource System Design Solar Field Collector and Receiver Financial Parameters Vind velocity adjustment 0 0.0066 Masorber tube inner diameter 0.007 0.0066 Masorber flow plug diameter 0.010 0.0285 0.011 0.0285 0.029 0.0205 0.021 Variation 1 Variation 1 Variation 1 Variation 1 Variation 1 0.0285 0.0385 0.041 0.055 0.01 0.025 0.0385 0.041 0.055 0.055	^
Location and Resource System Design Solar Field Collector and Receiver Financial Parameters Absorber tube inner diameter Absorber tube inner diameter 0.011 m Oligass envelope outer diareter Glass envelope outer diareter 0.115 m Variation 1 Variation 1 Variation 1 Variation 2 Variation 3 Variation 4* Variation 1 Variation 2 Variation 3 0.025 0.0385 0.045	
System Design Solar Field Collector and Receiver Financial Parameters Absorber tube inner diameter 0.007 m Absorber tube outer diameter 0.011 m Glass envelope inner diameter 0.115 m Absorber flow pattern Variation 1 Variation 2 Variation 3 Variation 4* Vision unitable flow pattern 0.115 m Absorber rube outer diameter 0.12 m Variation 1 Variation 2 Variation 3 Variation 4* 0.01 0.02	
Solar Field 0 0.672 0.002556 0 0 Collector and Receiver Collector and Receiver 1 0 0 0 0 0 Financial Parameters Absorber tube inner diameter 0.066 m Absorber flow plug diameter 0 0 0 0 Vind velocity adjustment 1 0 0 0 0 0 0 Steam temperature adjustment 1 0 0 0 0 0 0 Steam temperature adjustment 1 0 0 0 0 0 0	
Collector and Receiver Financial Parameters Wind velocity adjustment Absorber tube inner diameter 0.066 Absorber flow plug diameter 0.07 m Internal surface roughness 4.5e-005 Glass envelope inner diameter 0.115 m Absorber flow pattern Variation 1 Variation 2 Variation 3 Variation 4* Voi etweightige entries 0.985 0.01 0.996	
Collector and Receiver Financial Parameters -Evacuated tube heat loss model Absorber tube inner diameter 0.066 m Absorber flow plug diameter 0 m Absorber tube outer diameter 0.07 m Glass envelope inner diameter 0.115 m Glass envelope outer diameter 0.12 m Variation 1 Variation 2 Variation 3 Variation 4* Voient wighting ratio 0.985 0.01 0.005 0.985 0.966 0.986 0.965 0.985 0.965	
Financial Parameters Absorber tube inner diameter 0.066 m Absorber flow plug diameter 0 m Absorber tube outer diameter 0.07 m Internal surface roughness 4.5e-005 Glass envelope inner diameter 0.115 m Absorber flow pattern Tube flow Glass envelope outer diameter 0.12 m Absorber material type 304L Variation 1 Variation 2 Variation 3 Variation 4* Specify polynomial coefficients or use 0.96 0.96 0.8 0	
Absorber tube inner diameter 0.000 m Internal surface roughness 4.5e-005 Glass envelope inner diameter 0.115 m Absorber flow pattern Tube flow Glass envelope outer diameter 0.12 m Absorber material type 304L Variation 1 Variation 2 Variation 3 Variation 4* Variation 2 0.01 0.005 0 Specify polynomial coefficients or use 0.96 0.1384	
Glass envelope inner diameter 0.115 m Absorber flow pattern Tube flow Glass envelope outer diameter 0.12 m Absorber material type 304L Variation 1 Variation 2 Variation 3 Variation 4*	
Glass envelope outer dianceter 0.12 m Absorber material type 304L Variation 1 Variation 2 Variation 3 Variation 4* Vision two indications 0.985 0.01 0.005 0 Specify polynomial coefficients or use 0.966 0.965 Value 0.1384	
Variation 1 Variation 2 Variation 3 Variation 4* Variation 1 0.985 0.01 0.005 0 Specify polynomial coefficients or use 0.96 0.96 0.8 0	
Specify polynomial coefficients or use	
Specify polynomial coefficients or use	
0.1384 0.65 0.65 0.1384	- 1
physical model of evacuated tube.	
Envelope emittance 0.86 0.86 1 0	
Envelope transmittance 0.963 0.963 1 0	
Broken Glass Broken Glass Broken Glass Broken Glass	
Annulus gas type Air Air Air Air	
Annulus pressure (torr) 0.0001 750 750 0 Estimated avg. heat loss (W/m) 150 1100 1500 0	
Bellows shadowing 0.96 0.96 0.96 0	
Dirt on receiver 0.98 0.98 1 0	
Aggregate Weighted Losses	
Average field temp difference at design 204.5 'C	
Simulate > In Heat loss at design 244.317 W/m	
Parametrics Stochastic Receiver thermal derate 0.977567	
P50 / P90 Macros	
Collector optical loss at normal incidence 0.74613	

- If you have questions after the webinar or are watching a video recording, please contact us with questions:
 - o <u>https://sam.nrel.gov/support</u>
 - o <u>sam.support@nrel.gov</u>

Thank you!

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